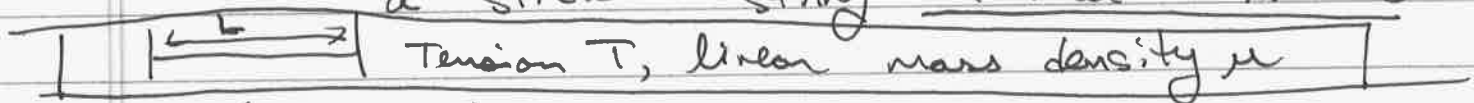


①

Lecture 10.1

Recall: we have been studying the motion of a stretched string fixed at both ends



Wave equation:

$$\frac{\partial^2 y(x,t)}{\partial x^2} = \frac{1}{v^2} \frac{\partial^2 y(x,t)}{\partial t^2}$$

$$\uparrow v^2 = \frac{T}{\mu} \text{ for string}$$

Solutions: (Normal modes)

$$y_n(x,t) = A_n \sin\left(\frac{2\pi x}{\lambda_n}\right) \cos(\omega_n t)$$
$$\lambda_n = \frac{2L}{n} \quad \omega_n = \frac{n\pi v}{L}$$

* Note Boundary conditions key in getting this form

(i.e. why no $\cos\left(\frac{2\pi x}{\lambda_n}\right)$?

Because $y(0,t) = 0$ (fixed end)

Motivates discussion of different boundary conditions

* These are key to determining the spectrum of modes!

②

→ Vibrations of air columns (tube of air w/ sound waves propagating inside)

→ When an end is open, ~~far~~ less pressure change due to motion of air molecules (pushing into "∞" space)

Air motion there is maximum

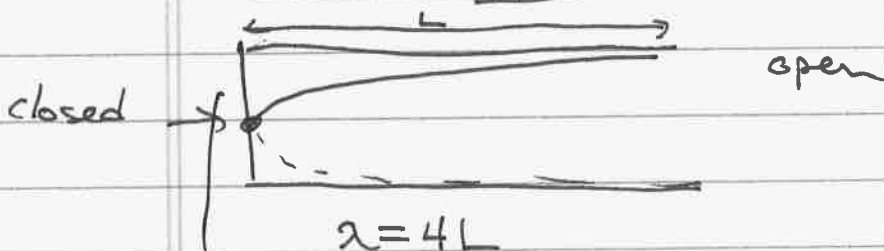
→ At closed end, air molecules cannot move (would leave vacuum behind)



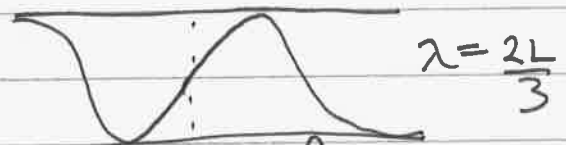
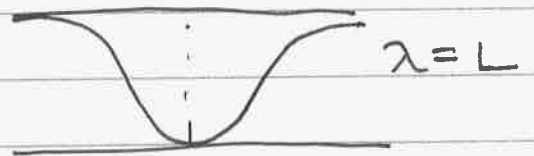
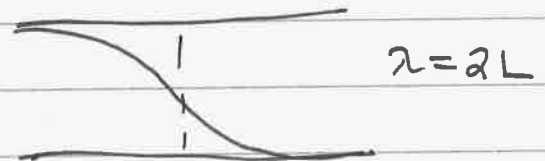
Summary

→ Node at closed end, anti-node at open end
(Plot horizontal disp. as function of x)

Closed / Open



Open / Open



General

$$\lambda = \frac{4L}{(2n-1)}$$

$$\lambda = \frac{4}{3}L$$

$$\lambda = \frac{4L}{5}$$

General

$$\lambda = \frac{2L}{n} = \frac{4L}{(2n)}$$

③

→ We know that a wave equation governs the sound waves

$$\frac{\partial^2 \xi(x,t)}{\partial x^2} = \frac{1}{v_{\text{sound}}^2} \frac{\partial^2 \xi(x,t)}{\partial t^2}$$

(ξ is horizontal displacement of air that is at position x when at equilibrium)

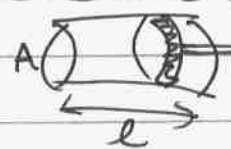
For string: $v^2 = \frac{T}{\mu}$

... what is analogous relation for sound in air or other materials?

→ Speed of sound

Modulus of gas: $K = -V \frac{dp}{dV}$ (analog of $\Delta p = -\frac{K}{V} \Delta V$
 $\sim \Delta F = -k \Delta x$)
↑
constant

→ Note temperature will change ^{with} ~~the~~ oscillation (adiabatic)

Say tube with area A , length l ;  piston
has gas at pressure p , density ρ

Ideal gas: $p = \frac{1}{3} \rho v_{\text{rms}}^2$ → NOT SOUND SPEED
↳ (v^2) average squared velocity

total mass $m = \rho l A \Rightarrow p = \frac{1}{3} \frac{m}{l A} v_{\text{rms}}^2$

or $p = \frac{2}{3} \frac{E_K}{l A}$ average KE of all gas in tube

→ Now consider moving the piston inward

④

Force on piston $F_p = Ap$

work done on gas: $\Delta W = -pA\Delta l$
↑ distance piston is moved

Assume only place energy goes is KE of gas

$$\Delta E_k = -pA\Delta l$$

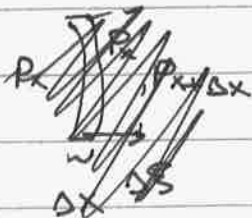
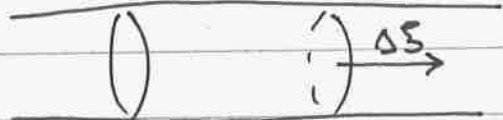
but $\Delta p = \Delta \left(\frac{2}{3A} \frac{E_k}{l} \right) = \frac{2}{3A} \left(\frac{\Delta E_k}{l} - E_k \frac{\Delta l}{l^2} \right)$

$$= \frac{2}{3A} \left(\frac{-pA\Delta l}{l} - \frac{3Alp}{2} \frac{\Delta l}{l^2} \right)$$

$$\Delta p = -\frac{5}{3} p \frac{\Delta l}{l} \rightarrow -\frac{5}{3} p \frac{\Delta V}{V}$$

rewrite:
$$-V \frac{\Delta p}{\Delta V} = \frac{5}{3} p \Rightarrow \left| K_{adiabatic} = \frac{5}{3} p \right|$$

Equation of motion:



$$\Delta P_x = -\frac{K}{V} \Delta V = -\frac{K(A\Delta s)}{A\Delta x} = -K \frac{\Delta s}{\Delta x}$$

$$\rightarrow \Delta P_x = -K \frac{\partial s}{\partial x}$$

$$F = A \Delta P_x - A \Delta P_{x+\Delta x} = -AK \left(\frac{\partial s}{\partial x} \Big|_x - \frac{\partial s}{\partial x} \Big|_{x+\Delta x} \right)$$

$$= +AK \frac{\partial^2 s}{\partial x^2} \Delta x$$

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→ Newton's 2nd law

$$F = AK \frac{\partial^2 \xi}{\partial x^2} \Delta x = \Delta m \frac{\partial^2 \xi}{\partial t^2} = \rho A \Delta x \frac{\partial^2 \xi}{\partial t^2}$$

$$\text{or } \boxed{\frac{\partial^2 \xi}{\partial x^2} = \frac{\rho}{K} \frac{\partial^2 \xi}{\partial t^2}}$$

$$\boxed{v_{\text{sound}}^2 = \frac{K}{\rho} = \frac{\gamma P}{\rho}} \quad (\approx 1.667 \frac{P}{\rho})$$

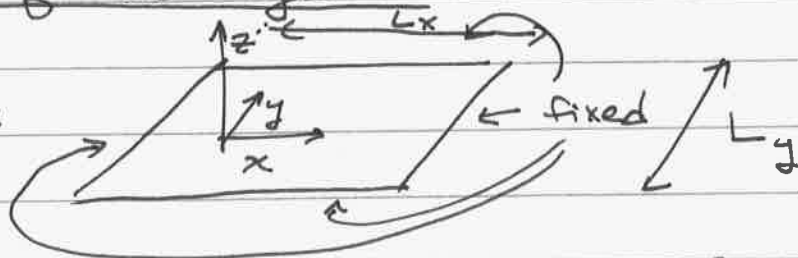
Note for many gases, relation between ρ and E_k is not so straightforward
 (energy of rotation of particles, vibrations of molecules, etc)

generally $K = \gamma P$ for nontrivial gases

air $\gamma = 1.4$ (compare w/ 1.667...)
 \uparrow N_2, O_2, CO_2, \dots

→ Normal modes of 2D system

A square drum:



$z = z(x, y, t)$

Wave equation: $\frac{\partial^2 z}{\partial x^2} + \frac{\partial^2 z}{\partial y^2} = \frac{1}{v^2} \frac{\partial^2 z}{\partial t^2}$

Solutions

$$z(x, y, t) = C_{n_1, n_2} \sin\left(\frac{n_1 \pi x}{L_x}\right) \sin\left(\frac{n_2 \pi y}{L_y}\right) \cos\left(\frac{\omega_{n_1, n_2} t}{v}\right)$$

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↙ surface tension (Force/length)

$$\omega_{n_1 n_2}^2 = \left(\frac{S}{\sigma} \right)^{\frac{1}{2}} \left[\left(\frac{n_1 \pi}{L_x} \right)^2 + \left(\frac{n_2 \pi}{L_y} \right)^2 \right] \sim \frac{v^2 n^2 \pi^2}{L^2}$$

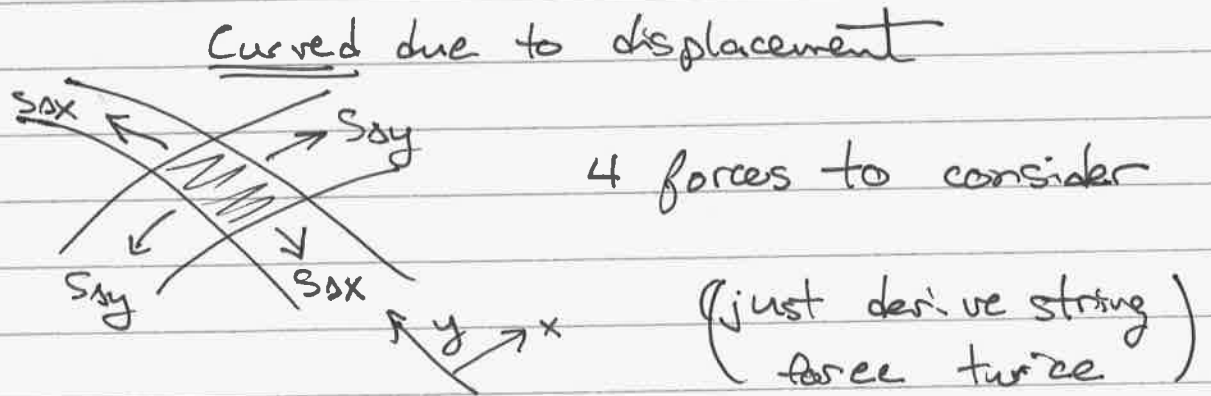
↑ mass per area

$$v^2 = \frac{S}{\sigma} \quad \left(\text{like } \frac{T}{\mu} \right) \rightarrow \text{Intuition from 1D waves on large plane}$$

↪ ~~can't get it~~

Why is wave equation $\sum_i \frac{\partial^2 \sigma}{\partial x_i^2} = \frac{1}{v^2} \frac{\partial^2 \sigma}{\partial t^2}$?

Consider path on surface:



$$F_{\text{net}} = S_{\Delta x} \Delta y \left(\frac{\partial^2 z}{\partial x^2} + \frac{\partial^2 z}{\partial y^2} \right) = \sigma \Delta x \Delta y \frac{\partial^2 z}{\partial t^2}$$

$$\text{or } \left[\frac{\partial^2 z}{\partial x^2} + \frac{\partial^2 z}{\partial y^2} = \left(\frac{\sigma}{S} \right) \frac{\partial^2 z}{\partial t^2} \right]$$

↓
1/v^2

7

Fourier Analysis

Back to 1D \rightarrow string fixed at both ends

Consider general amp. and phase of n 'th normal mode

$$y_n(x,t) = A_n \sin\left(\frac{n\pi x}{L}\right) \cos(\omega_n t - \delta_n)$$

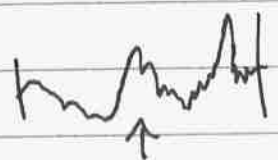
Total motion = superposition of all normal modes

$$y(x,t) = \sum_{n=1}^{\infty} A_n \sin\left(\frac{n\pi x}{L}\right) \cos(\omega_n t - \delta_n)$$

take picture (freeze frame) @ time $t = t_0$

$$y(x, t_0) = \sum_{n=1}^{\infty} \underbrace{A_n \cos(\omega_n t_0 - \delta_n)}_{\text{call this } B_n} \sin\left(\frac{n\pi x}{L}\right)$$

$$= \sum_{n=1}^{\infty} B_n \sin\left(\frac{n\pi x}{L}\right)$$


string configuration
at time t_0

\rightarrow Shape of string @ t_0 entirely given by coefficients B_n !